ELEC4410

Control System Design

Lecture 5: Reference & Disturbance Feedforward. Cascade Control

School of Electrical Engineering and Computer Science The University of Newcastle



Lecture 5: Reference & Disturbance Feedforward. Cascade Control - p. 1/21

Review Affine Parameterisation.



- Review Affine Parameterisation.
- Reference Feedforward.



- Review Affine Parameterisation.
- Reference Feedforward.
- Disturbance Feedforward.



- Review Affine Parameterisation.
- Reference Feedforward.
- Disturbance Feedforward.
- Cascade Control.



- Review Affine Parameterisation.
- Reference Feedforward.
- Disturbance Feedforward.
- Cascade Control.

Reference: Control System Design, Goodwin, Graebe & Salgado.



All sensitivity functions are affine in Q(s).

 $T_o(s) = Q(s)G_o(s)$ Complementary Sensitivity $S_o(s) = 1 - Q(s)G_o(s)$ Sensitivity $S_{io}(s) = (1 - Q(s)G_o(s))G_o(s)$ Input Disturbance Sensitivity $S_{uo}(s) = Q(s)$ Control Sensitivity

Unlike the case of C(s), which is nonlinear in the sensitivity functions, making it difficult to tune C(s) to achieve a desired closed loop performance i.e.

$$T_o(s) = \frac{G_o(s)C(s)}{1 + G_o(s)C(s)}$$



All sensitivity functions are affine in Q(s).

 $T_o(s) = Q(s)G_o(s)$ Complementary Sensitivity $S_o(s) = 1 - Q(s)G_o(s)$ Sensitivity $S_{io}(s) = (1 - Q(s)G_o(s))G_o(s)$ Input Disturbance Sensitivity $S_{uo}(s) = Q(s)$ Control Sensitivity

Unlike the case of C(s), which is nonlinear in the sensitivity functions, making it difficult to tune C(s) to achieve a desired closed loop performance i.e.

$$T_o(s) = \frac{G_o(s)C(s)}{1 + G_o(s)C(s)}$$

The nominal loop is internally stable if and only if Q(s) is a stable and proper transfer function and C(s) is

$$C(s) = \frac{Q(s)}{1 - Q(s)G_o(s)}$$



Affine Parameterisation in terms of **Q**





Affine Parameterisation in terms of Q



Affine Parameterisation in terms of C





By use of Q(s) we can shape 1 of 4 nominal sensitivities.

 $T_o(s) = Q(s)G_o(s)$ Complementary Sensitivity $S_o(s) = 1 - Q(s)G_o(s)$ Sensitivity $S_{io}(s) = (1 - Q(s)G_o(s))G_o(s)$ Input Disturbance Sensitivity $S_{uo}(s) = Q(s)$ Control Sensitivity



b By use of Q(s) we can shape 1 of 4 nominal sensitivities.

 $T_o(s) = Q(s)G_o(s)$ Complementary Sensitivity $S_o(s) = 1 - Q(s)G_o(s)$ Sensitivity $S_{io}(s) = (1 - Q(s)G_o(s))G_o(s)$ Input Disturbance Sensitivity $S_{uo}(s) = Q(s)$ Control Sensitivity

Some trade-offs with respect to bandwidth of the closed loop that need to be considered are: reference tracking (B.W ↑), measurement noise (B.W ↓), modelling errors (B.W ↓), output disturbance rejection (B.W ↑) and the controller output (B.W ↓).



b By use of Q(s) we can shape 1 of 4 nominal sensitivities.

 $T_o(s) = Q(s)G_o(s)$ Complementary Sensitivity $S_o(s) = 1 - Q(s)G_o(s)$ Sensitivity $S_{io}(s) = (1 - Q(s)G_o(s))G_o(s)$ Input Disturbance Sensitivity $S_{uo}(s) = Q(s)$ Control Sensitivity

- Some trade-offs with respect to bandwidth of the closed loop that need to be considered are: reference tracking (B.W ↑), measurement noise (B.W ↓), modelling errors (B.W ↓), output disturbance rejection (B.W ↑) and the controller output (B.W ↓).
- We know inversion is a key idea of control.



• One way to design Q(s) is

 $Q(s) = F_Q(s)[G_o(s)]^{-1}$

However, recall, it is not always possible to invert $G_o(s)$ exactly. Therefore use $G_o^i(s)$ which is a stable approximation to $[G_o(s)]^{-1}$

 $Q(s) = F_Q(s)G_o^i(s)$



• One way to design Q(s) is

 $Q(s) = F_Q(s)[G_o(s)]^{-1}$

However, recall, it is not always possible to invert $G_o(s)$ exactly. Therefore use $G_o^i(s)$ which is a stable approximation to $[G_o(s)]^{-1}$

 $Q(s) = F_Q(s)G_o^i(s)$

• Use $F_Q(s)$ to ensure properness of Q(s).



• One way to design Q(s) is

 $Q(s) = F_Q(s)[G_o(s)]^{-1}$

However, recall, it is not always possible to invert $G_o(s)$ exactly. Therefore use $G_o^i(s)$ which is a stable approximation to $[G_o(s)]^{-1}$

 $Q(s) = F_Q(s)G_o^i(s)$

- Use $F_Q(s)$ to ensure properness of Q(s).
- Note that the characteristic equation of F_Q(s) will also be the characteristic equation of T_o(s) (and of S_o(s)) if all the stable poles of G_o(s) are included in the approximate inversion.



What about S_{io}(s)? We can see that the poles of G_o(s) will appear in it. These poles will only be controllable from the disturbance!

 $S_{io}(s)=(1-Q(s)G_o(s))G_o(s)$



What about S_{io}(s)? We can see that the poles of G_o(s) will appear in it. These poles will only be controllable from the disturbance!

$$S_{io}(s) = (1 - Q(s)G_o(s))G_o(s)$$

What can we do about this? Slow poles in G_o(s) will cause a transient associated with an input disturbance to decay at a rate dictated by these modes. The fix, essentially adding zeros to S_o(s) at the location of the poles in G_o(s) to be cancelled.



What about time delay, e^{-sτ}? For small τ, use Padé approximation to model delay. Otherwise use Smith controller design, where Q(s) is based on the rational part of the model only.

$$G_o(s) = e^{-s\tau} \overline{G}_o(s)$$
 and
 $Q(s) = F_Q(s) \overline{G}_o^i(s)$



What about time delay, e^{-sτ}? For small τ, use Padé approximation to model delay. Otherwise use Smith controller design, where Q(s) is based on the rational part of the model only.

$$G_o(s) = e^{-s\tau} \overline{G}_o(s)$$
 and
 $Q(s) = F_Q(s) \overline{G}_o^i(s)$

Smith Controller in Q Parameterisation Form





A disadvantage of a one-degree-of-freedom control system is that the performance criteria that can be realised are limited.



- A disadvantage of a one-degree-of-freedom control system is that the performance criteria that can be realised are limited.
- For example: if the roots of the characteristic equation are selected to provide a certain amount of relative damping, the maximum overshoot of the step response may still be excessive, owing to the zeros in the closed loop transfer function.



- A disadvantage of a one-degree-of-freedom control system is that the performance criteria that can be realised are limited.
- For example: if the roots of the characteristic equation are selected to provide a certain amount of relative damping, the maximum overshoot of the step response may still be excessive, owing to the zeros in the closed loop transfer function.
- Reference Feedforward can help reduce this and other effects as we will see.



Reference Feedforward is also known as the 2nd degree-of-freedom control.



- Reference Feedforward is also known as the 2nd degree-of-freedom control.
- We can use a two-degree-of-freedom architecture to improve reference tracking.



- Reference Feedforward is also known as the 2nd degree-of-freedom control.
- We can use a two-degree-of-freedom architecture to improve reference tracking.
- Consider the two-degree-of-freedom architecture:





- Reference Feedforward is also known as the 2nd degree-of-freedom control.
- We can use a two-degree-of-freedom architecture to improve reference tracking.
- Consider the two-degree-of-freedom architecture:



The Feedforward control element is in the forward path of the feedback loop.



• The tracking performance can be quantified through the following equations (assuming the disturbances $D_i(s)$ and $D_o(s)$ are zero):

 $Y(s) = H(s)T_o(s)R(s)$ $U(s) = H(s)S_{uo}(s)R(s)$



• The tracking performance can be quantified through the following equations (assuming the disturbances $D_i(s)$ and $D_o(s)$ are zero):

 $Y(s) = H(s)T_o(s)R(s)$ $U(s) = H(s)S_{uo}(s)R(s)$





Another way of describing the flexibility of the two-degree-of-freedom controller is that the controller C(s) is usually designed to provide a certain degree of system stability and performance, but since the zeros of C(s) always become the zeros of the closed loop transfer function, unless some zeros are cancelled by the poles of the process, these zeros may cause a large overshoot in the system output even when the relative damping as determined by the characteristic equation is satisfactory.



- Another way of describing the flexibility of the two-degree-of-freedom controller is that the controller *C*(*s*) is usually designed to provide a certain degree of system stability and performance, but since the zeros of *C*(*s*) always become the zeros of the closed loop transfer function, unless some zeros are cancelled by the poles of the process, these zeros may cause a large overshoot in the system output even when the relative damping as determined by the characteristic equation is satisfactory.
- In this case and for other reasons, the transfer function H(s) may be used for the control or cancellation of the undesirable poles or zeros of the closed loop transfer function, while keeping the characteristic equation intact.



- Another way of describing the flexibility of the two-degree-of-freedom controller is that the controller *C*(*s*) is usually designed to provide a certain degree of system stability and performance, but since the zeros of *C*(*s*) always become the zeros of the closed loop transfer function, unless some zeros are cancelled by the poles of the process, these zeros may cause a large overshoot in the system output even when the relative damping as determined by the characteristic equation is satisfactory.
- In this case and for other reasons, the transfer function H(s) may be used for the control or cancellation of the undesirable poles or zeros of the closed loop transfer function, while keeping the characteristic equation intact.
- Of course we could introduce zeros in H(s) to cancel some of the undesirable poles of the closed loop transfer function that result from the controller C(s).



The key to the reference feedforward controller is that the controller H(s) is not in the loop of the system, so that it does not affect the roots of the characteristic equation of the original system.



- The key to the reference feedforward controller is that the controller H(s) is not in the loop of the system, so that it does not affect the roots of the characteristic equation of the original system.
- The poles and zeros of H(s) may be selected to add to or cancel poles and zeros of the closed loop transfer function, T_o(s).



- The key to the reference feedforward controller is that the controller H(s) is not in the loop of the system, so that it does not affect the roots of the characteristic equation of the original system.
- The poles and zeros of H(s) may be selected to add to or cancel poles and zeros of the closed loop transfer function, T_o(s).
- The essential idea of reference feedforward is to use H(s) to invert T_o(s) at certain key frequencies.



- The key to the reference feedforward controller is that the controller H(s) is not in the loop of the system, so that it does not affect the roots of the characteristic equation of the original system.
- The poles and zeros of H(s) may be selected to add to or cancel poles and zeros of the closed loop transfer function, T_o(s).
- The essential idea of reference feedforward is to use H(s) to invert T_o(s) at certain key frequencies.
- Note that, by this strategy, one can avoid using high gain feedback to bring $T_o(\omega_i)$ to 1.



- The key to the reference feedforward controller is that the controller H(s) is not in the loop of the system, so that it does not affect the roots of the characteristic equation of the original system.
- The poles and zeros of H(s) may be selected to add to or cancel poles and zeros of the closed loop transfer function, T_o(s).
- The essential idea of reference feedforward is to use H(s) to invert T_o(s) at certain key frequencies.
- Note that, by this strategy, one can avoid using high gain feedback to bring *T_o(ω_i)* to **1**.
- Note, however, that use of reference feedforward in this way does not provide perfect tracking if there is a change in the model.





Recall,

$$\frac{Y(s)}{R(s)} = H(s)T_o(s) \quad \text{and} \quad T_o(s) = \frac{G(s)C(s)}{1 + G(s)C(s)}$$





Recall,

$$\frac{Y(s)}{R(s)} = H(s)T_o(s) \quad \text{and} \quad T_o(s) = \frac{G(s)C(s)}{1 + G(s)C(s)}$$

Also,

$$\frac{U(s)}{R(s)} = H(s)S_{uo}(s)$$





Recall,

$$\frac{Y(s)}{R(s)} = H(s)T_o(s) \quad \text{and} \quad T_o(s) = \frac{G(s)C(s)}{1 + G(s)C(s)}$$

Also,

$$\frac{U(s)}{R(s)} = H(s)S_{uo}(s)$$

Can now shape the sensitivity from R(s) to Y(s) independent of the other sensitivities.



Performance of loop can be made robust by choosing B.W. of T_o(s) small.



- Performance of loop can be made robust by choosing B.W. of T_o(s) small.
- Again key idea \Rightarrow Inversion. However we now invert $T_o(s)$

 $H(s) = F_R(s)[T_o(s)]^{-1}$



- Performance of loop can be made robust by choosing B.W. of T_o(s) small.
- Again key idea \Rightarrow Inversion. However we now invert $T_o(s)$

 $H(s) = F_R(s)[T_o(s)]^{-1}$

▶ *H*(*s*) needs to be stable and proper.



- Performance of loop can be made robust by choosing B.W. of T_o(s) small.
- Again key idea \Rightarrow Inversion. However we now invert $T_o(s)$

 $H(s) = F_R(s)[T_o(s)]^{-1}$

- H(s) needs to be stable and proper.
- For regulators, which have constant set points, has no benefit.



- Performance of loop can be made robust by choosing B.W. of T_o(s) small.
- Again key idea \Rightarrow Inversion. However we now invert $T_o(s)$

 $H(s) = F_R(s)[T_o(s)]^{-1}$

- H(s) needs to be stable and proper.
- For regulators, which have constant set points, has no benefit.
- Good for set point tracking loops.



Disturbance Feedforward is also known as the 3rd degree-of-freedom control. Note that it only uses two-degrees-of-freedom in the control.



- Disturbance Feedforward is also known as the 3rd degree-of-freedom control. Note that it only uses two-degrees-of-freedom in the control.
- If a disturbance can be measured, then feedforward can be applied to improve disturbance rejection.



- Disturbance Feedforward is also known as the 3rd degree-of-freedom control. Note that it only uses two-degrees-of-freedom in the control.
- If a disturbance can be measured, then feedforward can be applied to improve disturbance rejection.
- Once again, a key concept is inversion.



- Disturbance Feedforward is also known as the 3rd degree-of-freedom control. Note that it only uses two-degrees-of-freedom in the control.
- If a disturbance can be measured, then feedforward can be applied to improve disturbance rejection.
- Once again, a key concept is inversion.
- We want to use the control signal, U(s), to cancel the disturbance, D_q(s), at the point where it enters the process.





Assuming zero reference,

 $Y(s) = S_o(s)G_{o2}(s)(1 + G_{o1}G_f(s))D_g(s).$



Assuming zero reference,

$$Y(s) = S_o(s)G_{o2}(s)(1 + G_{o1}G_f(s))D_g(s).$$

$$U(s) = -S_{uo}(s)(G_{o2}(s) + G_f(s))D_g(s).$$



Assuming zero reference,

$$Y(s) = S_o(s)G_{o2}(s)(1 + G_{o1}G_f(s))D_g(s).$$

Also,

$$U(s) = -S_{uo}(s)(G_{o2}(s) + G_{f}(s))D_{g}(s).$$

• $G_f(s)$ must be stable and proper (it is open loop control).



Assuming zero reference,

$$Y(s) = S_o(s)G_{o2}(s)(1 + G_{o1}G_f(s))D_g(s).$$

Also,

$$U(s) = -S_{uo}(s)(G_{o2}(s) + G_f(s))D_g(s).$$

- $G_f(s)$ must be stable and proper (it is open loop control).
- What should $G_f(s)$ be? To reject disturbances, i.e. Y(s) = 0, ideally

$$G_{o1}(s)G_f(s) = -1$$

$$\therefore G_f(s) = -[G_{o1}]^{-1}$$



Assuming zero reference,

$$Y(s) = S_o(s)G_{o2}(s)(1 + G_{o1}G_f(s))D_g(s).$$

Also,

$$U(s) = -S_{uo}(s)(G_{o2}(s) + G_{f}(s))D_{g}(s).$$

- $G_f(s)$ must be stable and proper (it is open loop control).
- What should $G_f(s)$ be? To reject disturbances, i.e. Y(s) = 0, ideally

```
G_{o1}(s)G_f(s) = -1
\therefore G_f(s) = -[G_{o1}]^{-1}
```

 G_f(s) would be expected to be high pass as G_{o1}(s) will typically possess a low pass characteristic. Therefore will have to include "fast" poles in G_f(s) to make proper.



Assuming zero reference,

$$Y(s) = S_o(s)G_{o2}(s)(1 + G_{o1}G_f(s))D_g(s).$$

Also,

$$U(s) = -S_{uo}(s)(G_{o2}(s) + G_f(s))D_g(s).$$

- $G_f(s)$ must be stable and proper (it is open loop control).
- What should $G_f(s)$ be? To reject disturbances, i.e. Y(s) = 0, ideally

```
G_{o1}(s)G_f(s) = -1
\therefore G_f(s) = -[G_{o1}]^{-1}
```

- G_f(s) would be expected to be high pass as G_{o1}(s) will typically possess a low pass characteristic. Therefore will have to include "fast" poles in G_f(s) to make proper.
- Gives more flexibility in the design as trade-offs can be relaxed.

If a measurement of a variable can be made between the point where a disturbance enters the process and the output of the process, then we can utilise feedback for disturbance rejection. This gives rise to "cascade control".





If a measurement of a variable can be made between the point where a disturbance enters the process and the output of the process, then we can utilise feedback for disturbance rejection. This gives rise to "cascade control".



Cascade control usually consists of two feedback loops



If a measurement of a variable can be made between the point where a disturbance enters the process and the output of the process, then we can utilise feedback for disturbance rejection. This gives rise to "cascade control".



Cascade control usually consists of two feedback loops

> Primary (outer) controlled by C_1 ,



If a measurement of a variable can be made between the point where a disturbance enters the process and the output of the process, then we can utilise feedback for disturbance rejection. This gives rise to "cascade control".



Cascade control usually consists of two feedback loops

- > Primary (outer) controlled by C_1 ,
- Secondary (inner) controlled by C_2 .



• $C_2(s)$ can be designed to attenuate $D_g(s)$ before it affects the output.



- $C_2(s)$ can be designed to attenuate $D_g(s)$ before it affects the output.
- Main benefits arise when





- $C_2(s)$ can be designed to attenuate $D_g(s)$ before it affects the output.
- Main benefits arise when
 - $G_a(s)$ contains nonlinearities that limit the loop performance





- $C_2(s)$ can be designed to attenuate $D_g(s)$ before it affects the output.
- Main benefits arise when
 - $G_a(s)$ contains nonlinearities that limit the loop performance
 - $G_b(s)$ is N.M.P. and / or contains time delays that limit B.W.





The output of the system is given by:

$$\begin{split} Y(s) &= C_2(s)G_o(s)S_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s)\\ G_o(s) &= G_{o1}(s)G_{o2}(s) \end{split}$$



The output of the system is given by:

$$\begin{split} Y(s) &= C_2(s)G_o(s)S_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s) \\ G_o(s) &= G_{o1}(s)G_{o2}(s) \end{split}$$

This can be re-written as:

$$Y(s) = G_b(s)T_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s)$$
(3)



The output of the system is given by:

$$\begin{split} Y(s) &= C_2(s)G_o(s)S_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s)\\ G_o(s) &= G_{o1}(s)G_{o2}(s) \end{split}$$

This can be re-written as:

$$Y(s) = G_b(s)T_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s)$$
(5)

If we don't have an inner loop, the output is given by

$$Y(s) = G_o(s)U(s) + G_{o2}(s)D_g(s)$$
 (6)



The output of the system is given by:

$$\begin{split} Y(s) &= C_2(s)G_o(s)S_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s)\\ G_o(s) &= G_{o1}(s)G_{o2}(s) \end{split}$$

This can be re-written as:

$$Y(s) = G_b(s)T_{o2}(s)U_1(s) + G_{o2}(s)S_{o2}(s)D_g(s)$$
(7)

If we don't have an inner loop, the output is given by

$$Y(s) = G_o(s)U(s) + G_{o2}(s)D_g(s)$$
 (8)

It can be seen in (1) that the disturbance will be somewhat attenuated when compared to (2).



> The secondary controller is usually designed first.



- > The secondary controller is usually designed first.
- > The primary controller is then designed based on an equivalent plant.

 $G_{eq} \triangleq G_b(s)T_{o2}(s)$



- > The secondary controller is usually designed first.
- > The primary controller is then designed based on an equivalent plant.

 $G_{eq} \triangleq G_b(s)T_{o2}(s)$

Generally, the secondary controller is designed such that

 $\mathsf{B.W} \text{ of } T_{o2}(s) > \mathsf{B.W} \text{ of } T_{o1}(s)$

